#### AMENDMENTS TO THE SPECIFICATION

# IN THE SPECIFICATION:

#### Page 1:

After the title, insert the following paragraph:

# CROSS-REFERENCE TO RELATED APPLICATION

This application is a division of Application No. 09/695,906 filed October 26, 2000.

### Page 1:

Please substitute the following paragraph for the paragraph beginning at line 6:

The present invention relates to an improvement in a toroidal type continuously variable transmission used as a transmission for a motor vehicle, for example, in order to realize a structure which can transmit great power and has sufficient endurance by adequately supplying oil to a power transmitting portion adequately at needs needed.

Please substitute the following paragraph for the paragraph beginning at line 14:

Application of a toroidal type continuously variable transmission as schematically shown in Figs. 8 and 9 to a

transmission of a motor vehicle has been investigated. For example, as disclosed in Japanese Utility Model Application Laid-Open No. 62-71465, in the toroidal type continuously variable transmission, an input side disc 2 is coaxially supported in coaxial with an input shaft 1, and output side disc 4 is secured to an end of an output shaft 3 coaxially disposed in coaxial with the input shaft 1. Within a casing containing the toroidal type continuously variable transmission, trunnions 6rocked, which rock around pivot shafts 5 located at positions twisted with respect to the input shaft 1 and the output shaft 3, are disposed between the input side disc 2 and the output side disc 4 in an axial direction.

#### Page 2:

Please substitute the following paragraph for the paragraph beginning at line 3:

That is to say, the trunnions 6 are provided at their outer surfaces with the pivot shafts 5 in , which are coaxial with each other. Further, proximal ends of displacement shafts 7 are supported on intermediate portions of the trunnions 6 so that inclination angles of

the displacement shafts 7 can be adjusted by rocking the trunnions 6 around the pivot shafts 5. Power rollers 8 are rotatably supported around the displacement shafts 7 supported by the trunnions 6. The power rollers 8 are pinched between opposed inner surfaces 2a and 4a of the input and output side discs 2 and 4. Each of the inner surfaces 2a, 4a has, at sections in section, a concave surface obtained by rotating an arc around the pivot shaft 5. Peripheral surfaces 8a of the power rollers 8 having spherical convex surfaces are contacted with the inner surfaces 2a, 4a.

#### Page 5:

Please substitute the following paragraph for the paragraph beginning at line 26:

Incidentally, the pair of displacement shafts 7 are disposed at positions diametrically opposed with respect to input shaft 15. Further, directions along which the pivot shaft portions 23 of the displacement shafts 7 are eccentric with respect to the support shaft portions 22 are the same directions (opposite directions in Fig. 11) with respect to the rotational directions of the input and

output side discs 2, 4. Further, the eccentric directions are substantially perpendicular to the installation direction of the input shaft 15. Accordingly, the power rollers 8 are supported for slight displacement in the axial direction (left-and-right direction in Fig. 10 and direction perpendicular to the plane of Fig. 11) of the input shaft 15. As a result, even if the power rollers 8 tends—tend—to be displaced in the axial direction of the input shaft 15 due to elastic deformation of the constructural parts caused by a great load acting on the constructural parts during the rotational force transmitting condition, such displacement can be absorbed without applying great or excessive force to the various parts.

#### Page 6:

Please substitute the following paragraph for the paragraph beginning at line 21:

Further, thrust ball bearings 26 and thrust needle bearings 27 are disposed between the outer surfaces of the power rollers 8 and the inner surfaces of the intermediate portions of the trunnions 6, and the thrust ball bearings

26 are located near the power rollers 8. The thrust ball bearings 26 serve to permit rotation of the power rollers 8 while supporting a thrust load acting on the power rollers 8. On the other hand, the thrust needle bearings 27 permit the pivot shaft portions 23 and outer races 28 of the thrust ball bearings 26 to be rocked around the support shaft portions 22 while supporting a thrust load acting on the outer races 28 from the power rollers 8.

#### Page 7:

Please substitute the following paragraph for the paragraph beginning at line 7:

Further, one—ends (left ends in Fig. 11) of the trunnions 6 are connected to drive rods 29, and drive pistons 30 are secured to outer peripheral surfaces of intermediate portions of the drive rods 29. The drive pistons 30 are mounted within drive cylinders 31 in an oiltight manner.

### Page 10:

Please substitute the following paragraph for the paragraph beginning at line 9:

Further, Japanese Patent Application Laid-Open No. 11-210855 discloses a lubricating device as shown in Figs. 12 and 13. In this conventional lubricating device, a lubricating post 33 is fixedly connected to, by a connecting screw 34, to a distal end of a support post 32 for supporting an intermediate portion of a support plate 20 for rocking movement and displacement movement in an axial direction (up-and-down direction in Figs. 12 and 13) of a-pivot shafts 5. Among four nozzle holes, having downstream ends opened at four equidistant locations on the circumference of a hold-down flange 35, formed on a distal end of the lubricating post 3334, downstream ends of two nozzle holes 36a shown in Fig. 13 are opened toward an inner surface 2a of an input side disc 2 and an inner surface 4a of an output side disc 4, respectively. On the other hand, downstream ends of two nozzle holes 36b shown in Fig. 12 are opened toward peripheral surfaces 8a of power rollers 8.

#### Page 15:

Please substitute the following paragraph for the paragraph beginning at line 2:

Similar to the above-mentioned conventional toroidal type continuously variable transmissions, a toroidal type continuously variable transmission according to the present invention comprises a housing, input and output side discs coaxially disposed in coaxial with each other and rotatably relative to each other within the housing, a plurality of trunnions rocked around respective pairs of coaxial pivot shafts located at positions twisted with respect to center lines of the input and the output side discs, displacement shaft supported for respective trunnions, and a plurality of power rollers rotatably supported on the displacement shafts and pinched between inner surfaces of the input and output side discs. The opposed inner surfaces of the input and output side discs have, at section, arc concave surfaces, and peripheral surfaces of the power rollers have spherical convex surfaces which are contacted with the inner surfaces. Further, traction oil can be supplied to contact areas between the inner surfaces of the discs and the peripheral surfaces of the power rollers.